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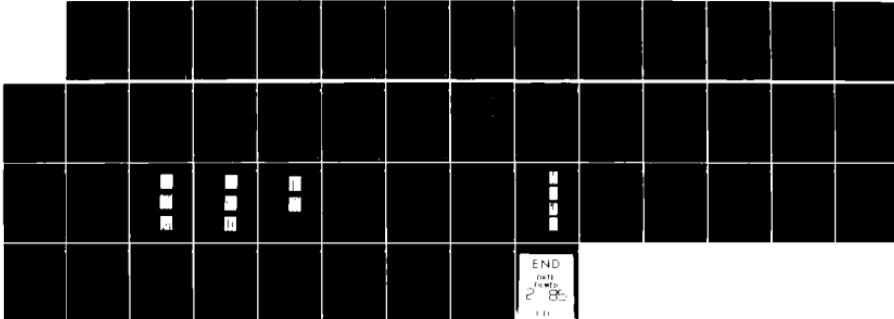
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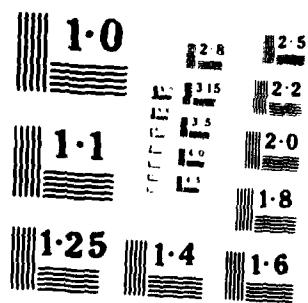
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**DEPARTMENT OF DEFENCE  
DEFENCE SCIENCE AND TECHNOLOGY ORGANISATION  
AERONAUTICAL RESEARCH LABORATORIES**

MELBOURNE, VICTORIA

STRUCTURES REPORT 399

**THE USE OF ADHESIVE-BONDED RIVETS TO LESSEN  
THE REDUCTIONS IN FATIGUE LIFE CAUSED BY  
RIVET HOLES**

by

J. Y. MANN, R. A. PELL, R. JONES and M. HELLER

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**SUMMARY**

Rivet holes are potential sites for fatigue crack initiation in aircraft structures. Several methods for improving the life of such details were investigated including coating the surface of the hole with adhesive, cold-expansion of the holes, the insertion of close-fit rivets and the use of adhesively-bonded rivets.

Of the various techniques examined only that involving adhesively-bonded rivets provided any significant improvements in fatigue life. It resulted in a reduction in fatigue crack propagation rate of about 50%, compared with that for specimens incorporating open holes.

A finite element analysis indicated that adhesive bonding significantly reduces both the local stress concentration at the hole and the stress intensities at the crack tips, thus retarding crack initiation and reducing fatigue crack propagation rates. However, the effective reduction in stress intensity resulting from bonding (about 17%) is much less than the 50% predicted by the finite element analysis. This discrepancy is attributed mainly to shortcomings in the model for defining the characteristics and behaviour of the adhesive.



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## 1. INTRODUCTION

During the full-scale fatigue testing of Mirage III fighter wings at the Swiss Federal Aircraft Factory (E + W), Switzerland, fatigue cracks were discovered at the innermost bolt holes along the rear flanges of the main spars. Subsequently, crack indications were confirmed at identical locations in wings of the Royal Australian Air Force Mirage III fleet (Ref. 1). As a consequence, several investigations were undertaken at the Aeronautical Research Laboratories (ARL) to explore methods for increasing the fatigue lives at critical sections of the spars (Refs 1-3).

An area in the wing main spar of particular concern was the first bolt hole in the lower rear flange. A detail of the spar in this region is shown in Fig. 1. The development of a life-enhancement scheme for this portion of the spar was complicated by the presence of two through-the-flange single-leg-anchor-nut (SLAN) rivet holes located close to the bolt hole in a chordwise direction. Although the installation of interference-fit steel bushes at the bolt hole virtually inhibited crack initiation at the bolt hole (Refs 1, 2) and the adoption of a modified system for securing the SLAN obviated the need for through-the-flange rivets, a consequence was that the SLAN rivet holes then became the critical locations for fatigue crack initiation. Several alternatives including reaming, cold-expanding and the insertion of close-fit rivets were tested as part of the main life-enhancement investigation for the spar, but these did not provide any significant improvements in life for cracks which initiated at the rivet holes.

Adhesive bonding of close-fit rivets in the SLAN holes was then proposed as a method of improving fatigue life. This was based on the premise that the adhesive would result in better load transfer through the rivet (and so doing reduce the stress concentrating effect of the hole) and, subsequent to crack initiation, reduce the stress intensity at the crack tip (Ref. 4). In addition, the adhesive could act as an environmental barrier (Ref. 5), or provide an interlayer which would reduce the effects of fretting between the rivet and hole surface. The proposal was investigated in two complementary series of fatigue tests which are covered by this report.

## 2. SPECIMENS AND TESTING PROGRAM

Figure 2 illustrates the basic forms of the two types of fatigue specimens employed in this investigation. The use of two types was necessitated by the availability at the time of suitable test material. Type (a) being made from offcuts of B.S. 1168<sup>\*</sup> aluminium alloy extruded bar 63.5 mm x 31.75 mm in section (Serial GR) used for the investigation reported in Reference 6, while Type (b) were taken from offcuts of 32 mm thick 2014-1651 aluminium alloy rolled plate (Serial GH) covered by Reference 3. In both cases the axis of the specimen was parallel to the direction of extrusion/rolling of the material. The chemical compositions, tensile properties and fracture toughness of the materials are given in Appendix 1.

### 2.1 Type (a) specimens (total 21)

In these specimens, the pitch between the twin holes was the same as the nominal pitch of the two rivet holes in the SLAN, and the distance from the centre line of each hole to the closest side of the specimen was that from the inner side of the 8 mm spar bolt hole to the centre line of the first rivet hole i.e. 8.20 mm. A specimen thickness of 30 mm was chosen to correspond to the thickness of the spar flange at the SLAN section.

<sup>\*</sup> The chemical compositions and static properties of these materials are equivalent to those of the French alloy A-L45G used for the manufacture of the spars.

Eight different hole treatments were investigated.

- (i) Holes drilled 3.3 mm diameter, 0.125 inch diameter universal head rivets inserted against a packing piece, with the tail of the rivet peened into the countersink. This generally represented the original condition in the RAAF Mirage IIIO wings.
- (ii) Holes drilled 3.3 mm diameter and left empty. A condition equivalent to that of rivet removal from the spar flange without any reworking of the rivet holes.
- (iii) Holes reamed to 4 mm diameter and left empty. Representing a situation in which the rivet holes were simply cleaned up for inspection.
- (iv) Holes as in (iii) incorporating selected 5.32 inch (4 mm) diameter countersink-head 2117 aluminium alloy rivets pressed-in by hand to provide a neat fit. Designed to allow some load transfer through the rivet but to enable easy rivet removal for hole inspection during testing if required.
- (v) Holes and rivets as in (iv), but with rivets permanently bonded in position using an epoxy adhesive Type K 138.\*
- (vi) Holes cold-expanded using the Boeing split-sleeve process (Ref. 7) to finish at 4 mm diameter. Holes left empty. The cold-expanding process introduces a residual compressive stress field adjacent to the hole which can retard fatigue crack initiation and growth.
- (vii) Holes as in (vi) but incorporating close-fit rivets as in (iv).
- (viii) Holes and rivets as in (vii) but with rivets adhesively bonded as in (v).

## 2.2 Type (b) specimens (total 9)

In these specimens the distance from the centre line of the hole to the side of the specimen (8.70 mm) corresponded to that in the Type (a) specimens. The specimen thickness was, however, slightly less i.e. 2.8 mm. Four hole treatments were investigated.

- (i) Holes reamed to 4 mm diameter and left empty. Equivalent to Type (am).
- (ii) Holes reamed as in (i) and left empty, but a coating of adhesive applied to the hole surface.
- (iii) Holes reamed as in (i) but incorporating pressed-in close-fit rivets. Equivalent to Type (amv).
- (iv) Holes reamed as in (i) but incorporating adhesive-bonded close-fit rivets. Equivalent to Type (av).

## 3. FATIGUE TESTS

The multi-load-level fatigue testing sequence adopted for this investigation was identical to that used for the other Mirage life-enhancement programs (Refs 1-3, 6). It consisted of a 100-flight sequence of four different flight types as indicated in Fig. 3. Cycles of +6.5 g/ -1.5 g and +7.5 g/ -2.5 g (a total of 39 cycles in 100 flights) were applied at a cyclic frequency of 1 Hz, whereas the remaining 1950 cycles per 100 flights were at 3 Hz. Sine-wave loading was adopted throughout. All fatigue tests were carried out in a Tinius-Olsen servo-controlled electro-hydraulic fatigue machine, the 100-flight sequence being achieved using an EMR Model 1641 programmable function generator controlled by a punched tape.

\* Details of the hole preparation, etc. for the adhesive-bonding and cold-expansion processes are given in Appendix 2.

For Type (a) specimens fatigue loads were based on the assumption that +7.5 g corresponded to a gross-area stress of 235 MPa (34 100 psi) and that there was a linear stress/g relationship i.e. the 1 g gross-area stress was 31.3 MPa (4547 psi). This magnitude of stress was chosen on the basis of results from a previous investigation (Ref. 6) using the same batch of material so that individual fatigue specimens should have test durations of between about one and two days. Specimens with 3.3 mm diameter holes had a nominal nett area of 562 mm<sup>2</sup>, while those with 4 mm holes a nett area of 520 mm<sup>2</sup>. The resulting nett area stresses were 318 MPa (46 100 psi) and 344 MPa (49 900 psi) respectively - a difference of about 8%.

The nett area stress chosen for the Type (b) specimens was the same as that for Type (a) specimens incorporating 4 mm holes, i.e. 344 MPa (49 900 psi). In this case the gross area stress was 265 MPa (38 390 psi).

Tables 1 and 2 give the individual fatigue lives, log. average lives and standard deviations of log. life for the various groups of Type (a) and Type (b) specimens. The extent of the fatigue cracking and representative fractures for Type (a) specimens are illustrated in Figs 4 and 5 respectively, while similar information relating to Type (b) specimens is shown in Figs 6 and 7.

#### 4. DISCUSSION

The individual fatigue test series involving the twin hole and single hole specimens both demonstrate the effectiveness of adhesive-bonded rivets in providing a significant increase in the life to failure relative to those for other hole treatments. Compared with specimens having reamed open holes, the ratio of lives for specimens incorporating adhesive-bonded rivets in reamed holes are 2.65 and 2.61 for the twin-hole and single-hole specimens respectively. On the limited data available, the cold-expansion of the holes in the twin-hole specimens does not result in a significant increase in life compared with that of reamed open-hole specimens, but bonding of rivets in cold-expanded holes again provides a marked increase in life. Furthermore, an adhesive coating on the hole surface of the single-hole specimens has not resulted in an improvement in life.

For both types of specimens the lives of those incorporating pressed-in rivets were not significantly different to those of open-hole specimens with similar hole conditions. On the basis of nett area stresses, the 4 mm reamed open twin-hole specimens would have been expected (Ref. 6) to have an average life of about 55%, that of the 3.3 mm drilled hole specimens, i.e. 4200 flights. The greater actual life (5929 flights) of the reamed hole specimens probably reflects the much better hole surface finish in these compared with the drilled-hole specimens.

Three reasons which could be advanced for the significant improvement in life associated with the use of adhesive-bonded rivets are:

- (a) the adhesive acting as a barrier to inhibit crack initiation which might otherwise have been accelerated by environmental interaction;
- (b) the adhesive acting as a non-metallic interlayer, thus separating the rivets and hole surface and reducing the potentially deleterious effects of fretting;
- (c) the adhesive providing improved load transfer characteristics at the section, both before and after crack initiation.

The tests on single-hole specimens suggest that the adhesive coating, as such, does not play a major part in the increased life associated with adhesive-bonded rivets. An examination of the fracture surfaces of specimens with filled holes (i.e. incorporating either pressed-in or adhesive-bonded rivets) indicated the presence of fretting at or close to the countersink-end of nearly every hole, with lesser or no fretting at the other end of the holes. Of the 20 'filled-hole' specimens (15 twin-hole and five single-hole) the primary crack initiation in 13 was some distance from the ends of the hole and in the other four close to the end opposite to the countersink.

It was only in the other three cases (one twin-hole cold-expanded, adhesive-bonded specimen No. GR22B, and two single-hole, one each with pressed-in and adhesive-bonded rivets, GJ1Z and GJ1ZA respectively) that the primary crack developed from close to the countersink and fretting apparently did play a significant part in its initiation. Thus, as fretting was not a major factor in crack initiation in the non-adhesive-bonded specimens, the anti-fretting properties of the adhesive interlayer are unlikely to be responsible for the benefits resulting from the use of adhesive-bonded rivets.

An improved load transfer at the section containing the holes would be expected to delay the initiation of fatigue cracks by reducing the effective stress concentration and, providing the continuity of the adhesive was maintained, to reduce the fatigue crack propagation rate because of a reduction in the stress intensity at the crack tip. In order to elucidate this matter fractographic crack growth studies were made on several single-hole specimens (these being chosen in preference to the twin-hole specimens because of their less complicated crack development and to avoid problems associated with crack interactions which were apparent when two holes were present), and a complementary finite-element analysis made of the stress distributions around uncracked and cracked open holes and holes containing adhesively-bonded rivets.

One specimen from each of the four single-hole types was selected for detailed fractographic examination, and the respective fracture surfaces are illustrated in Fig. 7. It is clear that there are significant differences in crack development in these four specimens, and although a basis for selection was that of approximately equal maximum crack depths on both sides of the hole this criterion could not be satisfied in the case of the specimen with a pressed-in rivet (Type (bui)). The fracture surface feature used as the reference for crack growth measurements was that produced by the 7.5 g load which occurs only once in the 100-flight sequence, during flight 42. In every case growth data were obtained for the cracks initiating at both sides of the hole.

Details of the fractographic techniques employed are given in Appendix 3. These included the use of macrophotographs, an optical stereo microscope and a metallographic microscope. Crack growth measurements were obtained at distances as close as 0.020 to 0.039 mm from the hole surface in the case of specimens with open holes and pressed-in rivets and 0.125 mm in the case of the specimens with adhesive-bonded rivets. The incremental crack growth data obtained using the three techniques were combined to provide the series of crack growth plots illustrated in Fig. 8, and the curves (for the longer crack in each of the four specimens) shown in Fig. 9.

These curves show that the fatigue crack propagation period covers a much greater part of the life in adhesively-bonded rivet specimens than in the other three types of specimens. However, as the maximum crack depth for a given crack geometry is defined by the fracture toughness of the material, no great differences in the crack growth characteristics between the four types of specimens would have been anticipated at crack depths approaching complete fracture. Because of the absence of definable features on the fracture surfaces of adhesive-bonded specimens at crack depths of less than 0.125 mm, the fractographic studies did not provide any evidence relating to the effects of adhesive bonding on fatigue crack initiation in these specimens.

The fractographic crack growth measurements also allowed the determination of the incremental crack propagation rates at the different crack depths corresponding to the applications of the 7.5 g load. Two specimens were selected for detailed analysis. They were the open-hole specimen No. GJ1ZB and the adhesively-bonded rivet specimen No. GJ2B16, each of which had reasonably symmetric crack growth from both sides of the hole. This allowed the crack growth data for both sides of the holes in the individual specimens to be pooled. The crack depth versus incremental crack growth data were plotted on linear scales and second order polynomial curves fitted. (It should be noted however that there was considerable variability in the raw incremental growth data for crack depths of greater than about 2 mm). Calculated crack growth rates at selected crack depths are given in Table 3.

On the premise that (for similar crack geometries) any given crack propagation rate represents conditions of equivalent stress intensity factors in the two types of specimens, it can be seen from Table 3 that 0.5, 1.0, 1.5 and 2.0 mm deep cracks in adhesively-bonded rivet specimens are equivalent to those of 0.2, 0.6, 1.0 and 1.4 mm in depth respectively in open-hole specimens. Clearly, the percentage reduction in crack propagation rate associated with the use of bonded rivets decreases as the cracks become deeper, and would approach zero at final fracture.

conditions. Using data for a similar material published elsewhere (Ref. 9) it was estimated that a reduction in fatigue crack propagation rate of 50% should represent a reduction in stress intensity at the crack tip of about 17%. Reductions of about 50% in crack propagation rate are evident for the open-hole specimen when comparing the rates at crack depths of 2.0 and 1.1 mm, and 1.0 and 0.5 mm, and for the adhesive-bonded rivet specimen at depths of 2.2 and 1.3 mm, and 1.2 and 0.7 mm. At a crack depth of 0.5 mm the crack growth rate in the bonded-rivet specimen is about half that in the open hole specimen.

In order to understand the mechanisms which resulted in the increase in fatigue life of the adhesively-bonded specimen a detailed finite element analysis (Ref. 10) was undertaken on the single-hole specimen containing a bonded rivet. Initially the specimen was considered to be uncracked and, due to symmetry, only a quarter of it was modelled. The resultant plane strain finite element model consisted of 38 eight-noded isoparametric quadrilateral elements and eight six-noded isoparametric triangular elements (see Fig. 10(a)). Element stiffness matrices were computed using reduced integration and double precision and the solution was also performed using double precision.

The aluminium alloy rivet and specimen were both assumed to have a Modulus of Elasticity of 73 000 MPa and a Poisson's ratio of 0.32, while the adhesive was assumed to have a Modulus of Elasticity of 700 MPa and a Poisson's ratio of 0.35. The adhesive layer was very thin and although its thickness was not precisely known it was, for the purpose of this analysis, taken to be 0.0127 mm. It was assumed that the applied tensile stress corresponded to the maximum load of 7.5 g in the fatigue sequence. This resulted in a gross area 'applied' stress of 265 MPa and a nett section stress at the hole of 344 MPa.

Unfortunately no information on the static tensile or fatigue strength of the adhesive was available and so two separate analyses were undertaken where:

- (i) the adhesive was assumed to not yield or fail,
- (ii) the adhesive was assumed to fail wherever the peel stress was tensile — this represented a condition by which at least 50% of the glue line had failed.

The results of these analyses can be found in Table 4 as can the result for the case when the hole was unfilled. This shows that in each case the adhesively-bonded rivet has dramatically reduced the stresses around the hole and thus a significant increase in the life to crack initiation could be expected.

Attention is now directed to the situation when the hole is cracked. Cracks of various discrete depths, up to a maximum of 2.5 mm, were considered for the following cases:

- (i) a through crack on one side of the hole only (half the structure modelled),
- (ii) through cracks of equal depth, on each side of the hole (a quarter of the structure modelled).

The finite element model for these problems consisted of approximately 74 eight-noded isoparametric quadrilateral elements and 32 six-noded isoparametric triangular elements for case (i) and 36 and 24, respectively for case (ii) (see Fig. 10(b)). As before, reduced integration was used and the solution was performed in double precision. The results of this analysis are given in Tables 5 and 6. Also given are the values of the stress intensity factor for the case when the hole is unfilled and for the case when there is a crack but no rivet hole. The results for the double-sided crack case are shown plotted in Fig. 11.

There are a number of important points indicated by this analysis:

- (i) A bonded rivet significantly reduces the stress intensity factor, even after a significant proportion of the adhesive has failed. Thus the use of adhesive-bonded rivets would be expected to result in reduced fatigue-crack propagation rates.
- (ii) The values of stress intensity factors for the case of a crack on one side and for the case of a crack of the same depth on both sides of the hole are almost identical.

- (iii) As the crack depth increases a crack at a bonded rivet hole behaves as if the specimen does not contain a hole. In the present case this asymptotic behaviour is effectively reached at a crack depth of approximately 2.5 mm. This is particularly important since it allows simple, and yet accurate, analytical estimates to be obtained for stress intensity factors of cracked holes which are to be repaired by a bonded insert.
- (iv) In the case of a crack on both sides of the hole the value of the stress intensity factor for a 3.0 mm crack at a bonded rivet hole (with adhesive failed in tension) is approximately the same as for a 0.5 mm crack at an unfilled rivet hole.

Whilst this summary of the finite element analysis has concentrated on bonded rivets with a particular adhesive thickness, the more detailed investigation presented in Ref. 10 covers the effects of variable adhesive thickness and the use of a bonded sleeve in larger fastener bolts.

A comparison of the results of the finite element analyses for cracked specimens and the data obtained from the fractographic analysis indicates that the finite element analysis successfully predicts the observed trends. Referring to the data in Table 3 and Fig. 11 for the open hole and adhesively-bonded rivet (adhesive failed) specimens, a 50% reduction in observed fatigue crack propagation rates between pairs of crack depths corresponds to about 17% reduction in the stress intensity factors for both types of specimens. This value of 17% is in excellent agreement with that obtained from Ref. 9. However, the effective reduction in stress intensity at any given crack depth as a result of the insertion of an adhesively-bonded rivet is much less than that predicted by the finite element analysis. For example, at a crack depth of 0.5 mm the 50% reduction in crack propagation rate would suggest a reduction in stress intensity of about 17%, whereas the finite element analysis predicts a reduction of greater than 50%.

Clearly the finite element model for determining stress intensities incorporated a number of simplifying assumptions of which the geometry of the crack front relative to those which developed in actual fatigue specimens and the characteristics of the glue line are probably the most significant. If one assumes that there are fewer uncertainties in representing the situation for an unfilled hole, then on a relative basis the major reason for the discrepancies between actual and predicted stress intensities for the open-hole specimens and those incorporating adhesively-bonded rivets is in the adequacy of the model for defining the properties and behaviour of the adhesive. Nevertheless, the findings outlined in this Report suggest that the concept of adhesively-bonded inserts is worthy of a more detailed study.

## 5. CONCLUSIONS

1. The insertion of close-fit adhesively-bonded rivets in holes can provide significant improvements in fatigue life compared with that for specimens having open holes.
2. For small crack depths the rates of fatigue crack propagation in adhesively-bonded rivet specimens are only about half those in open-hole specimens.
3. Finite element analyses indicate that adhesive bonding significantly reduces both the local stress concentration at the hole and the stress intensities at the crack tips, thus retarding crack initiation and reducing fatigue crack propagation rates.
4. The effective reduction in stress intensity resulting from bonding (about 17%) is much less than the 50% predicted by the finite element analysis. This discrepancy is attributed mainly to shortcomings in the model for defining the characteristics and behaviour of the adhesive.

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**APPENDIX 1**  
**Properties of Test Materials**

*(a) Chemical composition (%)*

Element	British Standard 1168-1978	Test Material GR	QQ-A-255a (2014)	Test Material GJ
Cu	3.9-5.0	4.29	3.9-5.0	4.48
Mg	0.2-0.8	0.43	0.2-0.8	0.60
Mn	0.4-1.2	0.76	0.4-1.2	0.91
Fe	0.5 max	0.23	1.0 max	0.45
Si	0.5-0.9	0.74	0.5-1.2	0.86
Cr	0.15 max	not analyzed	0.1 max	0.04
Cr	0.10 max	0.01	0.1 max	0.04
Zn	0.25 max	0.20	0.25 max	0.12

*(b) Static tensile properties*

Property	British Standard 1168-1978	Test material GR	QQ-A-255a (2014-T651) (transverse)	Test material (transverse)	Test material (longitudinal)
0.1% proof stress (MPa)		466		458	470
0.2% proof stress (MPa)	440	474	407 (59 000 psi)	466	475
Ultimate tensile stress (MPa)	490	524	462 (67 000 psi)	506	509
Elongation (%)	5	11	4	8.5	9.3
0.1% proof stress/ultimate		0.89		0.91	0.92

*(c) Fracture toughness (K<sub>IC</sub>)*

	Test material GR*	Test material GJ†
MPa m <sup>0.5</sup>	32.0	25.0
ksi in <sup>-0.5</sup>	29.2	22.8

\* Average of five tests on 19 mm thick compact tension specimens.

† Average of five tests on 25 mm thick compact tension specimens.

## APPENDIX 2

### Hole Treatments

The holes in all specimens were initially drilled at 3.3 mm diameter (from the Datum Face), and countersunk 120°, 5.5 mm diameter at the Datum Face. Final reaming for all except Type (a) and (an) specimens was 4.010-4.028 mm diameter, also from the Datum Face.

#### *A. Boeing split sleeve cold-expansion*

Starting hole size	3.683/3.708 mm
Finished hole size (after reaming)	4.010-4.028 mm
Degree of cold-expansion	2.7 to 3.4%
Direction of expansion	Toward Datum Face

(Elastic recovery of the test specimen material occurs after withdrawal from the hole of the cold-expansion tool. The degree of cold-expansion is defined as the percentage difference in diameter between the starting hole size, and the maximum mandrel diameter plus twice the sleeve thickness.)

#### *B. Adhesive bonding*

The surface treatment of the holes consisted of a thorough degreasing using Methyl Ethyl Ketone followed by light abrasion using a stainless steel wire brush. Adhesive type K138, a two-part epoxy manufactured by CIBA, was then applied to the surfaces of the hole and the rivet, and the rivet worked into the hole to ensure that the surfaces of both the hole and the rivet were fully coated. The adhesive was then cured overnight at 40°C.

### APPENDIX 3

#### Fractographic Crack Growth Measurements

Each fracture was examined using three independent techniques:

- (i) a macrophotograph at about  $\times 15$  magnification to provide crack growth data at relatively large crack depths.
- (ii) an optical stereo microscope with diffuse illumination and magnifications of  $\times 100$ ,  $50$  and  $25$ , and
- (iii) a metallographic microscope with polarized vertical illumination, using magnifications of  $\times 500$  or  $50$ .

Both microscopes were fitted with a cross-hair in one eye piece. The specimens were mounted on an X-Y stage fitted with photo-electrical digital micrometers reading to 0.001 mm, and the crack depths corresponding to the applications of the 7.5 g load were accurately measured by traversing the specimens under the microscope objectives. Whenever possible the traverse of the fracture surface was perpendicular to the axis of the hole.

Because of the substantial overlapping of the region of the fracture by each independent visualization technique about 80% of the crack depth data were duplicated. However, measurements obtained using the metallographic microscope were considered to be the most accurate at short crack depths ( $\times 500$ ) and large crack depths ( $\times 50$ ), and those using the stereo microscope the most accurate at intermediate crack depths. Thus the crack depth versus life data presented are not a simple average of the measurements obtained by the different techniques but represent the actual measurements obtained using the technique considered to be most accurate over particular regions of the fracture. When combined together they provided a coherent set of data covering the entire line of traverse.

As a check on the validity of the interpretation of the individual fracture markings on each specimen all the data obtained for a particular fracture using the three independent techniques were combined and examined on the basis of a relationship between crack depth and incremental crack growth. For intermediate and large crack depths (e.g. for greater than about 0.5 mm) this relationship was found to be approximately linear. On the relatively rare occasions when individual measurements indicated wide departures from this relationship the particular data were reassessed. When data could not be validated by independent measurements using more than one technique (about 20% of the data), the validity of individual measurements in such cases were assessed by the individual relationship between crack depth and incremental crack growth.

TABLE I  
Fatigue test results, Type (a) specimens

Hole treatment	Specimen No. GR	Life (flights)
(i) Drilled 3.3 mm, filled 1.8 inch rivets	18F 13B 19C	9,542 10,042 10,320
		log. average life s.d. log. life
		9,963 0.017
(ii) Drilled 3.3 mm, open holes	26B 16I	7,500 8,105
		log. average life s.d. log. life
		7,797 0.024
(iii) Reamed 4 mm, open holes	25B 18B	5,542 6,342
		log. average life s.d. log. life
		5,929 0.041
(iv) Reamed 4 mm, pressed-in 5.32 inch rivets	17B 23B 14B	3,735 3,842 6,335
		log. average life s.d. log. life
		4,496 0.129
(v) Reamed 4 mm, adhesive-bonded 5.32 inch rivets	20B 13C 16B	14,642 16,040 16,440
		log. average life s.d. log. life
		15,688 0.026
(vi) Expanded 4 mm, open holes	19B 24B	7,142 9,442
		log. average life s.d. log. life
		8,212 0.086
(vii) Expanded 4 mm, pressed-in 5.32 inch rivets	17C 14I 21B	5,942 7,642 8,080
		log. average life s.d. log. life
		7,159 0.071
(viii) Expanded 4 mm, adhesive-bonded 5.32 inch rivets	15B 22B 15C	10,542 23,542 26,130
		log. average life s.d. log. life
		18,648 0.216

TABLE 2  
Fatigue test results, Type (b) specimens

Hole treatment	Specimen No. GJ	Life (flights)
(i) Reamed 4 mm, open holes	IU	2,542
	IZB	3,242
	log. average life	2,871
	s.d. log. life	0.075
(ii) Reamed 4 mm, open holes coated with adhesive	IY	2,742
	IV	2,842
	log. average life	2,792
	s.d. log. life	0.011
(iii) Reamed 4 mm, pressed-in rivet	2A16	3,542
	IZ	3,820
	log. average life	3,678
	s.d. log. life	0.023
(iv) Reamed 4 mm, adhesive-bonded rivet	1X	5,742
	1ZA	7,942
	2B16	8,942
	log. average life	7,415
	s.d. log. life	0.100

TABLE 3  
Crack Propagation Rates Determined from Fractographic Measurements

Crack depth (mm)	Crack propagation rate (mm/100 flights)		Reduction in rate by bonding (%) [(I - B A) × 100]
	Open hole specimen GJ1ZB (A)	Bonded rivet specimen GJ2B16 (B)	
0.1	0.026	0.0001	99.6
0.2	0.038	0.009	76
0.3	0.052	0.019	63
0.4	0.065	0.028	57
0.5	0.080	0.039	51
0.6	0.094	0.049	48
0.7	0.109	0.060	45
0.8	0.125	0.071	43
0.9	0.141	0.082	42
1.0	0.158	0.094	41
1.1	0.175	0.106	39
1.2	0.193	0.119	38
1.3	0.211	0.131	38
1.4	0.229	0.144	37
1.5	0.248	0.158	36
1.6	0.268	0.172	36
1.7	0.288	0.186	35
1.8	0.308	0.200	35
1.9	0.329	0.215	35
2.0	0.351	0.230	35
2.1	0.373	0.245	34
2.2	0.395	0.261	34
2.3	0.418	0.277	34
2.4	0.441	0.293	34
2.5	0.465	0.310	33

TABLE 4  
Maximum principal stress for the uncracked specimen

Case considered	Stress (MPa)
Unfilled hole	853
Bonded rivet (no adhesive failure)	337
Bonded rivet (adhesive failed in tension)	427

TABLE 5  
Stress Intensity Factors  $K$  (MPa  $m^{1/2}$ ) for a cracked hole—crack on one side only

Crack depth (mm)	Bonded rivet		No hole	Open hole
	No adhesive failure	Adhesive failed in tension		
0.5	9.1	12.1		25.6
0.9	11.3	15.0		27.8
1.5	14.3	18.4	13.1	29.9
2.0	15.9	20.1	14.8	30.9
2.3	17.1	21.4	16.0	32.1

TABLE 6  
Stress Intensity Factors  $K$  (MPa  $m^{1/2}$ ) for a cracked hole—for a crack on  
both sides

Crack depth (mm)	Bonded rivet		Open hole
	No adhesive failure	Adhesive failed in tension	
0.5	9.1	12.2	26.4
0.9	11.3	15.3	29.7
1.5	14.5	18.8	33.9
2.3	17.6	22.6	39.2
2.5		23.6	

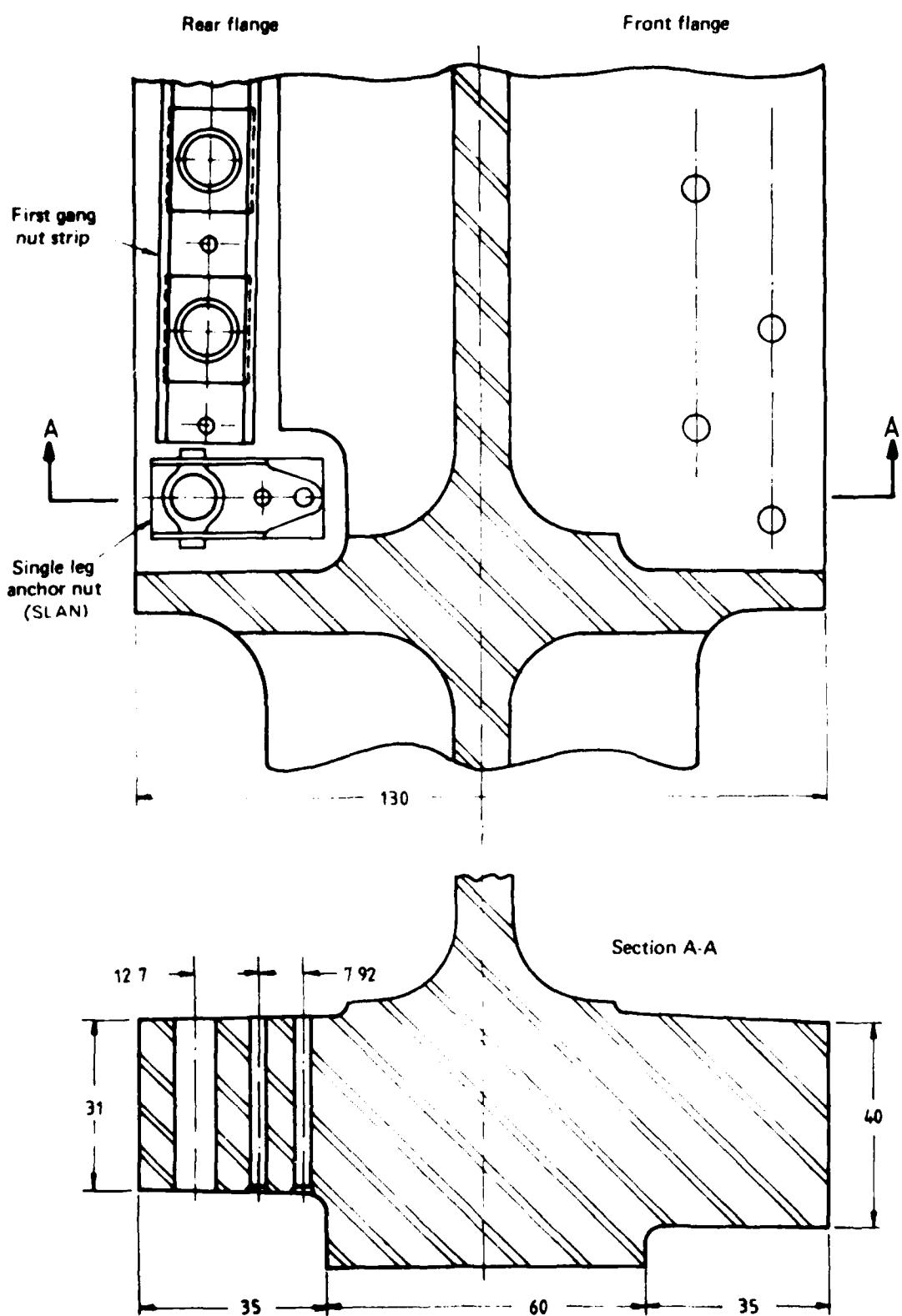
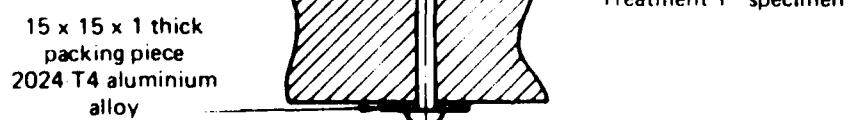
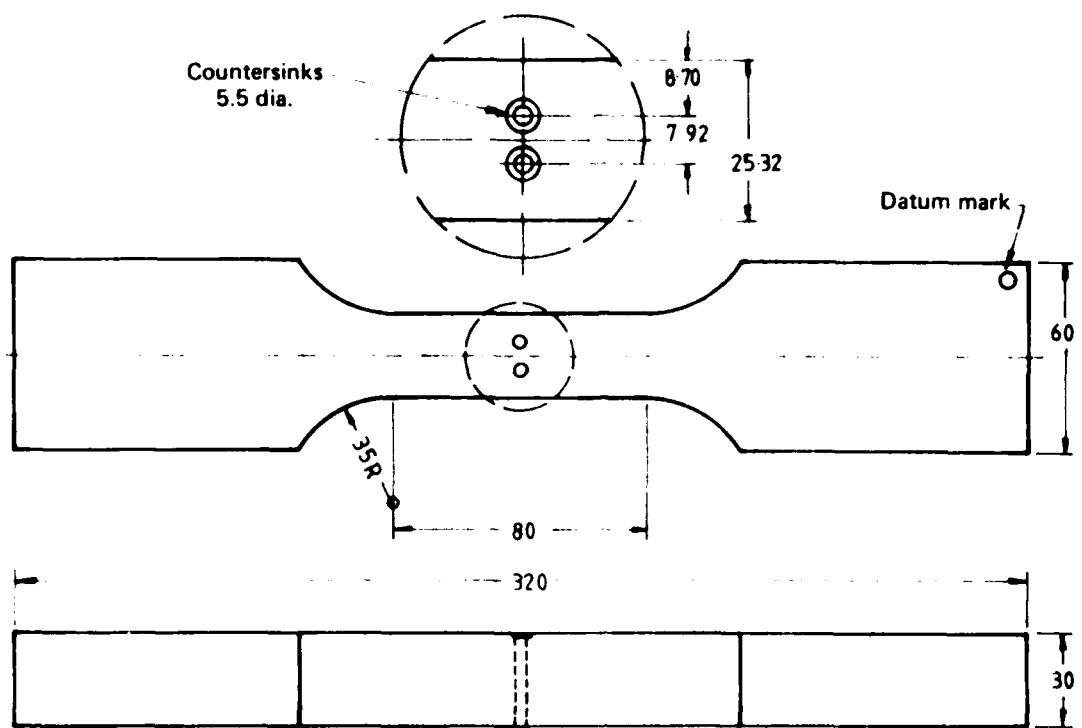
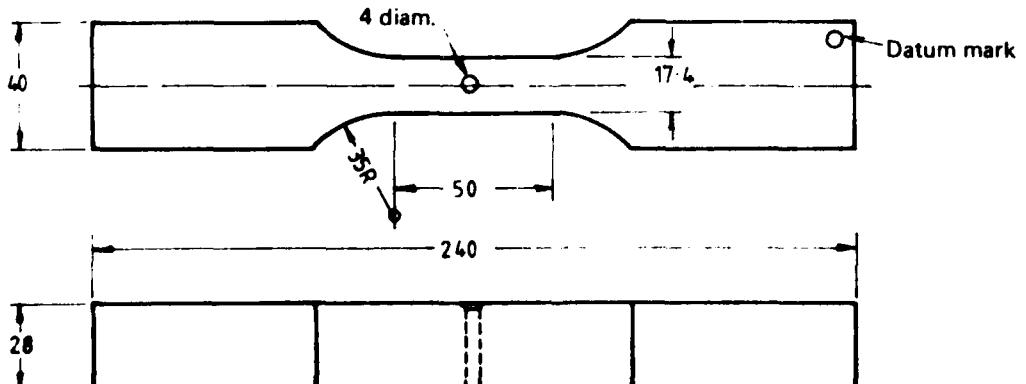


FIG. 1 MIRAGE 1110 SPAR - LOWER SURFACE AT SLAN SECTION

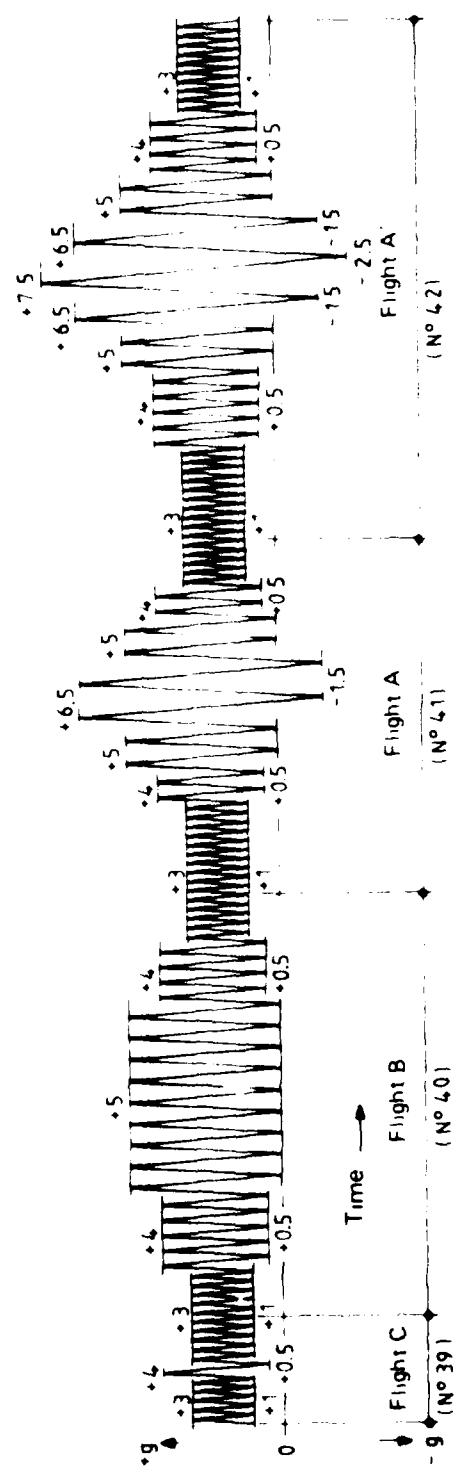
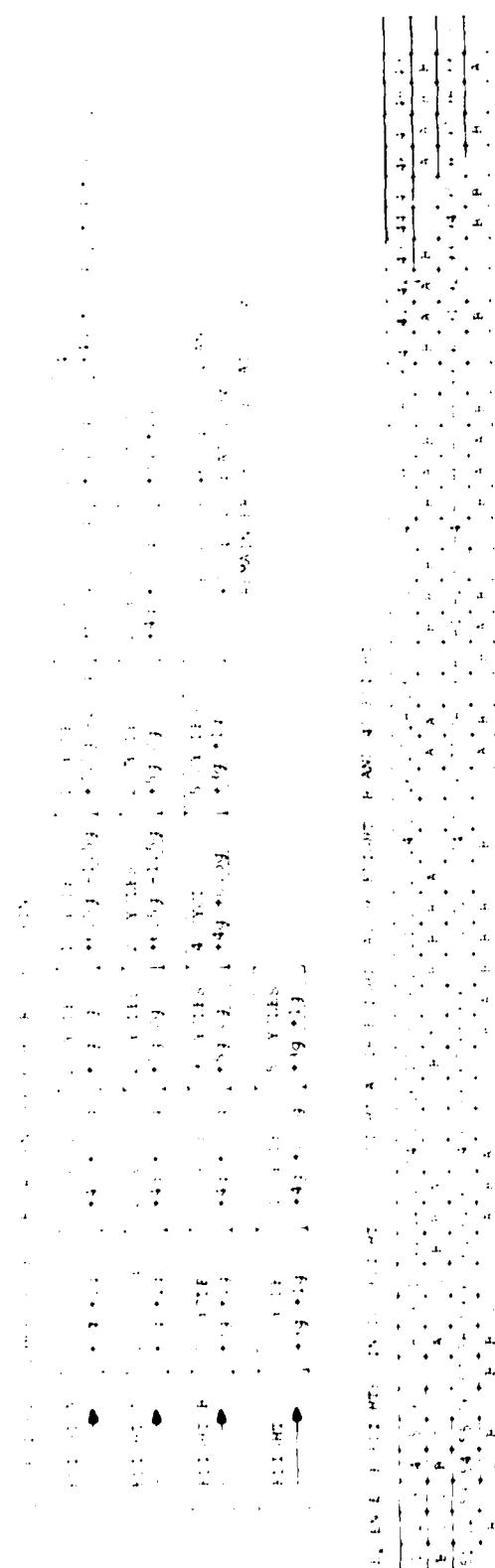


(a) Twin SLAN rivet hole specimen

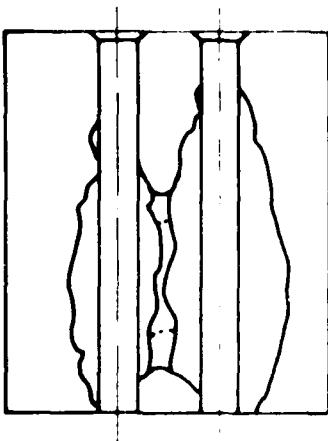


(b) Single SLAN rivet hole specimen

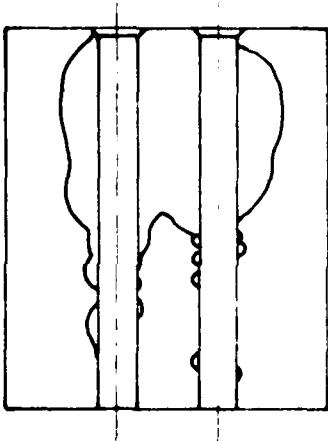
FIG. 2 FATIGUE TEST SPECIMENS  
(all dimensions in mm)



Specimen No. GR18E  
Flights 9,542



Specimen No. GR138  
Flights 10,042



Specimen No. GR19C  
Flights 10,320

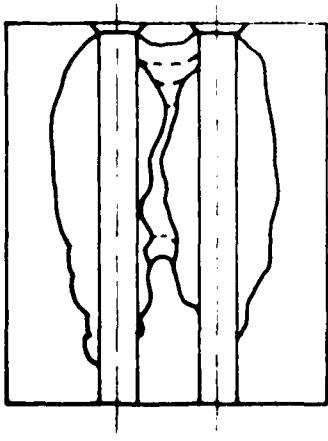
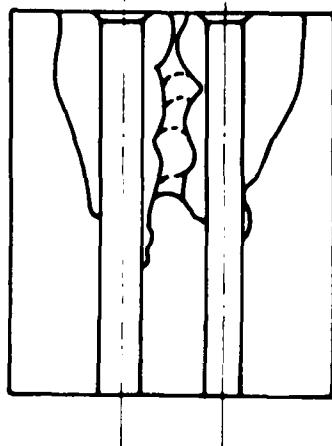


FIG 4 (ii) FRACTURES TYPE (ai) SPECIMENS. DRILLED 3.3 mm, FILLED 0.125 in. RIVETS

Specimen No. GR26B  
Flights 7,500



Specimen No. GR16E  
Flights 8,105

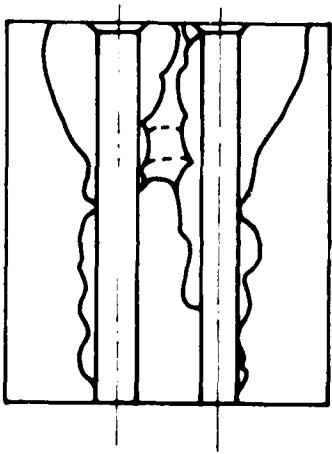
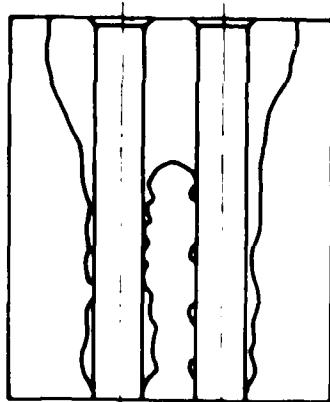


FIG. 4 (ii) FRACTURES TYPE (ai) SPECIMENS. DRILLED 3.3 mm, OPEN HOLES

Specimen No. GR258  
Flights 5,542



Specimen No. GR188  
Flights 6,342

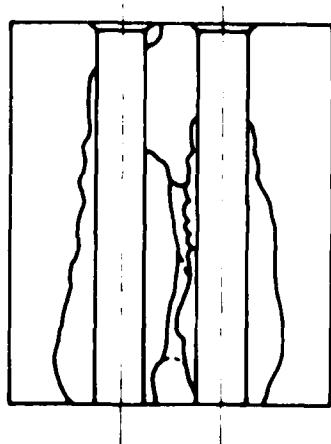
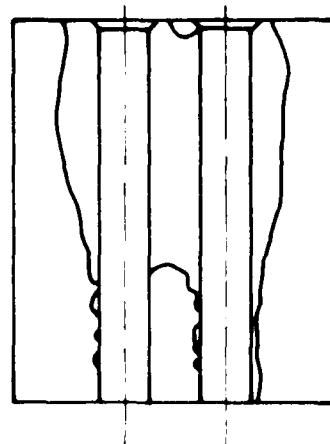
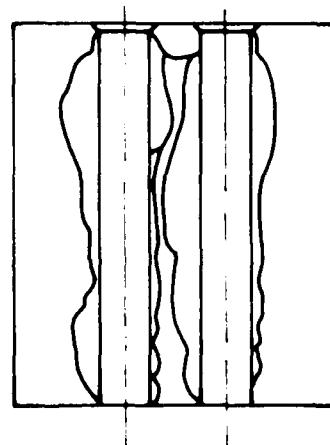


FIG. 4 (iii) FRACTURES TYPE (iii) SPECIMENS. REAMED 4 mm, OPEN HOLES

Specimen No. GR17B  
Flights 3,735



Specimen No. GR23B  
Flights 3,842



Specimen No. GR14B  
Flights 6,335

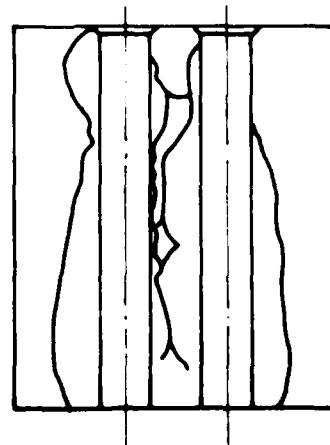
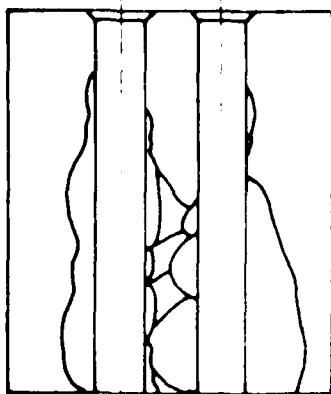
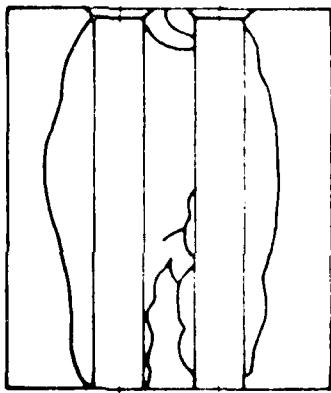


FIG. 4 (iv) FRACTURES TYPE (aiv) SPECIMENS. REAMED 4 mm,  
PRESSED-IN  $\frac{1}{16}$  in. RIVETS

Specimen No. GR20B  
Flights 14,642



Specimen No. GR13C  
Flights 16,040



Specimen No. GR16B  
Flights 16,440

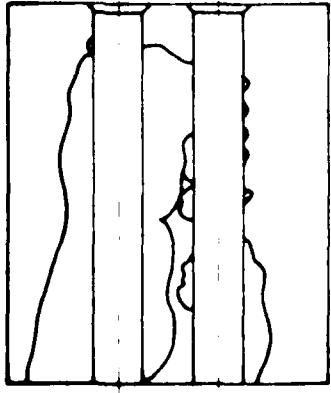
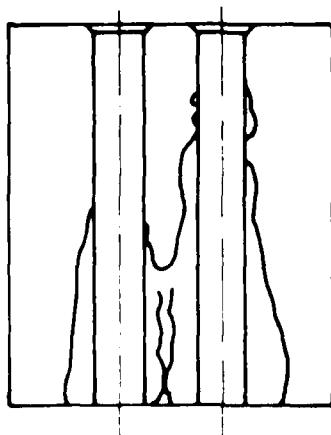


FIG. 4 (v) FRACTURES TYPE (av) SPECIMENS. REAMED 4 min, ADHESIVE BONDED  
1/2 in. RIVETS

Specimen No. GR198  
Flights 7,142



Specimen No. GR24B  
Flights 9,442

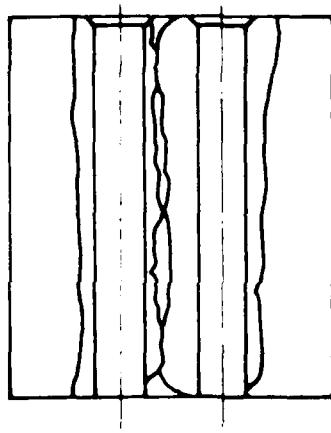
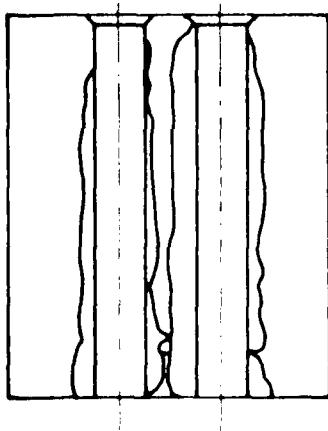
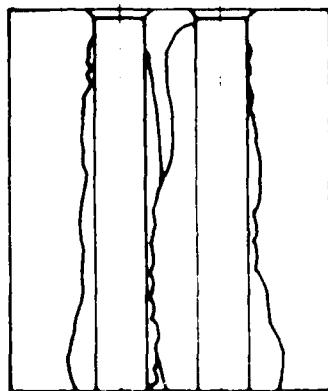


FIG. 4 (vi) FRACTURES TYPE (avi) SPECIMENS. COLD EXPANDED 4 mm, OPEN HOLES

Specimen No. GR17C  
Flights 5,942



Specimen No. GR14E  
Flights 7,642



Specimen No. GR21B  
Flights 8,080

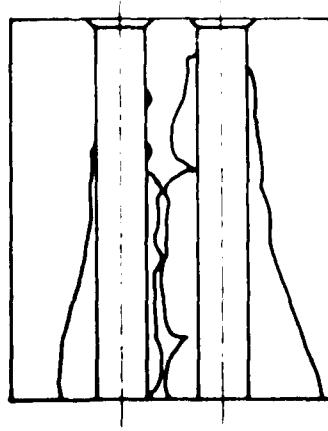
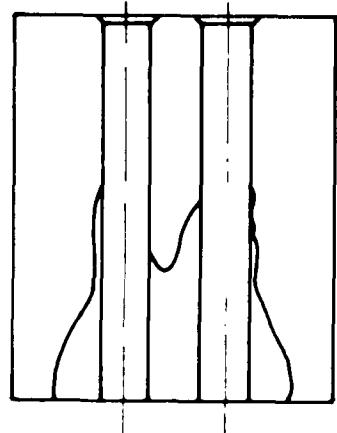
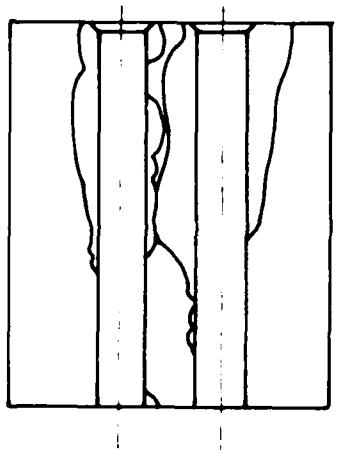


FIG. 4 (vii) FRACTURES TYPE (avii) SPECIMENS. COLD EXPANDED 4 mm, PRESSED-IN  
1/16 in. RIVETS

Specimen No. GR15B  
Flights 10,542



Specimen No. GR22B  
Flights 23,542



Specimen No. GR15C  
Flights 26,130

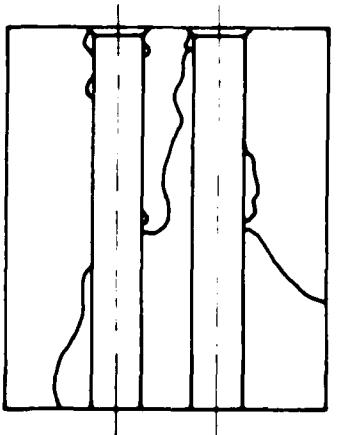
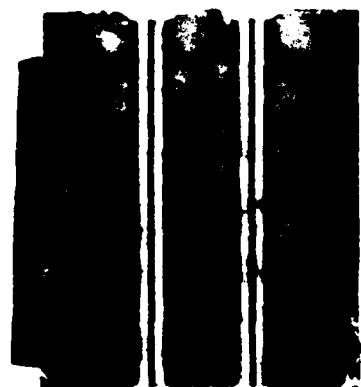


FIG. 4 (viii) FRACTURES TYPE (aviii) SPECIMENS. COLD EXPANDED 4 mm,  
ADHESIVE BONDED  $\frac{1}{16}$  in. RIVETS

Specimen No. GR18E  
Type (ai) — 3.3 mm  
drilled holes, 0.125  
in. rivets



Specimen No. GR16E  
Type (aii) — 3.3 mm  
drilled holes, open



Specimen No. GR18B  
Type (aiii) — 4 mm  
reamed holes, open



FIG. 5 (a) FRACTURE SURFACES TYPE (a) SPECIMENS

Specimen No. GR14B  
Type (av) - 4 mm  
reamed holes, pressed  
in rivets



Specimen No. GR13C  
Type (av) - 4 mm  
reamed holes, adhesive  
bonded rivets



Specimen No. GR24B  
Type (avi) - 4 mm  
cold-expanded holes,  
open



FIG. 5 (b) FRACTURE SURFACES TYPE (a) SPECIMENS

Specimen No. GR14E  
Type (avii) - 4 mm  
cold expanded holes,  
pressed-in rivets

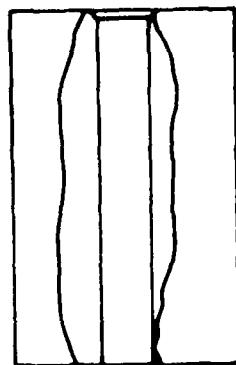


Specimen No. GR22B  
Type (aviii) - 4 mm  
cold expanded holes,  
adhesive-bonded  
rivets



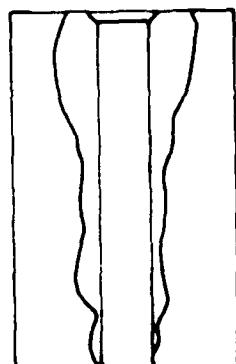
FIG. 5 (c) FRACTURE SURFACES TYPE (a) SPECIMENS

Specimen No. GJ1U  
Flights 2,542

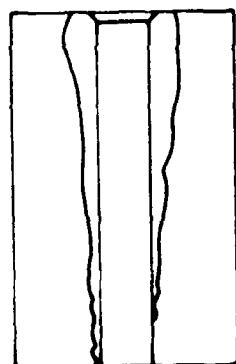


(i)

Specimen No. GJ1ZB  
Flights 3,242



Specimen No. GJ1Y  
Flights 2,742



(ii)

Specimen No. GJ1V  
Flights 2,842

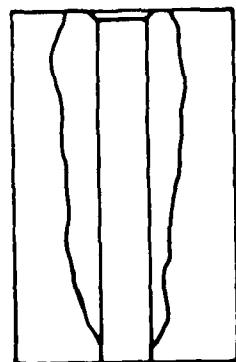
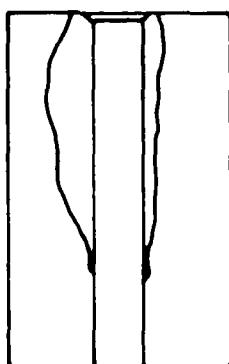


FIG. 6 FRACTURES TYPE (b) SPECIMENS.  
(i) Reamed 4 mm, open holes, and  
(ii) Reamed 4 mm, open holes adhesive coated

Specimen No. GJ2A18  
Flights 3,542



Specimen No. GJ1Z  
Flights 3,820

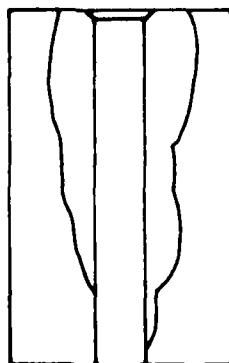
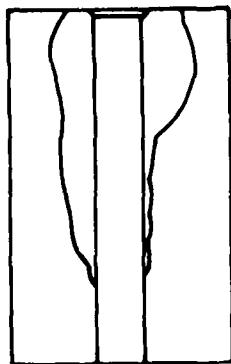
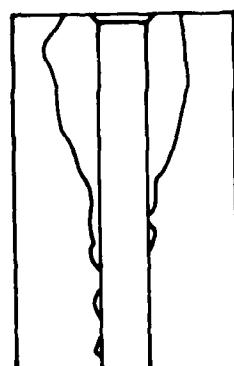


FIG. 6 (iii) FRACTURES TYPE (biii) SPECIMENS.  
Reamed 4 mm, pressed-in rivet

Specimen No. GJ1X  
Flights 5,742



Specimen No. GJ1ZA  
Flights 7,942



Specimen No. GJ2B16  
Flights 8,942

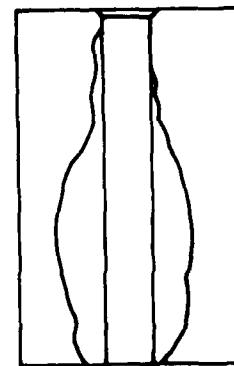
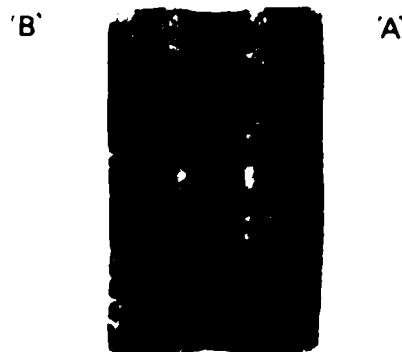


FIG. 6 (iv) FRACTURES TYPE (biv) SPECIMENS.  
Reamed 4 mm, adhesive-bonded rivet

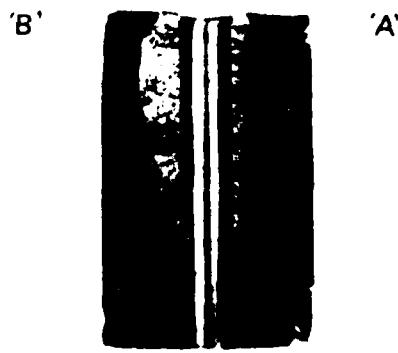
Specimen No. GJ1ZB  
Type (bi) - 4 mm  
reamed hole, open



Specimen No. GJ1Y  
Type (bii) - 4 mm  
reamed hole, adhesive  
coated, open



Specimen No. GJ2A16  
Type (biii) - 4 mm  
reamed hole, pressed-in  
rivet



Specimen No. GJ2B16  
Type (biv) - 4 mm  
reamed hole, adhesive  
bonded rivet

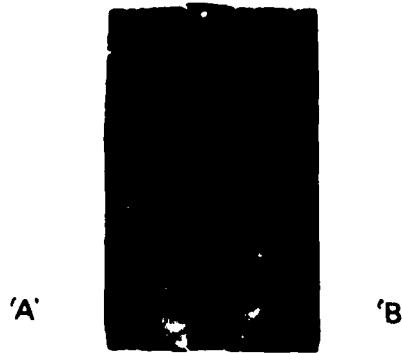


FIG. 7 FRACTURE SURFACES TYPE (b) SPECIMENS

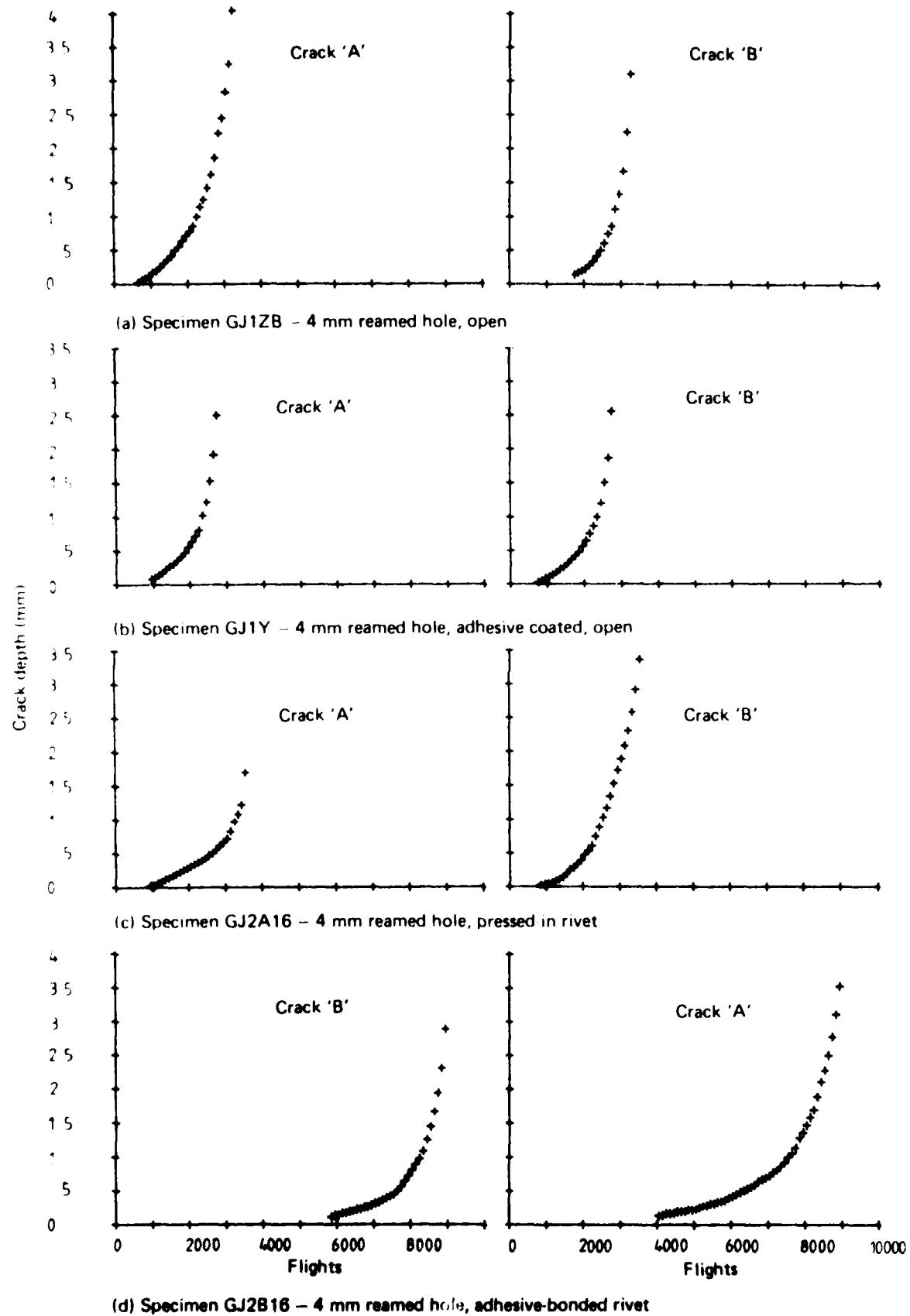
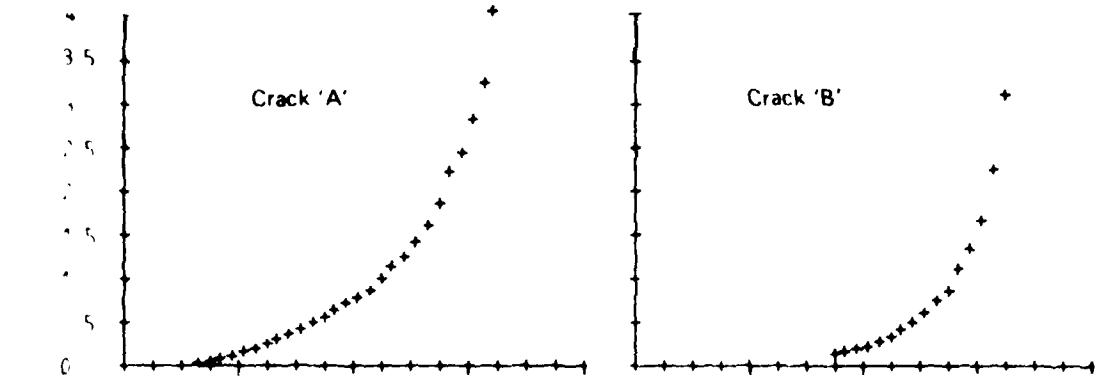
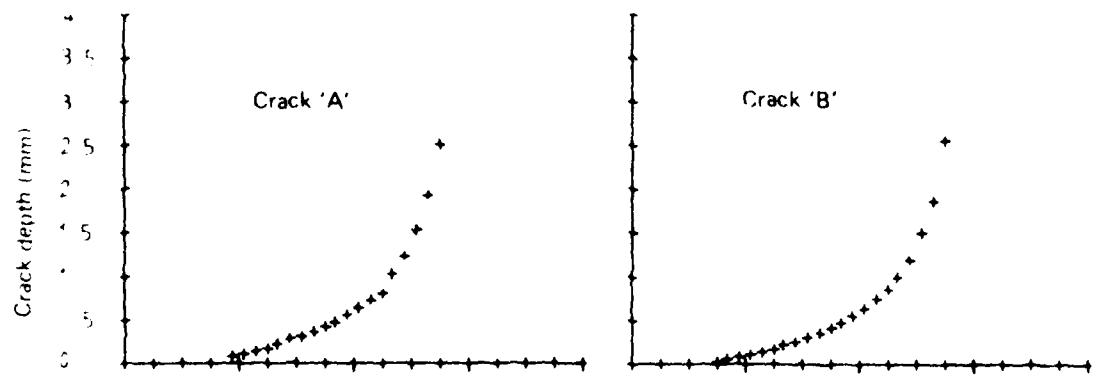


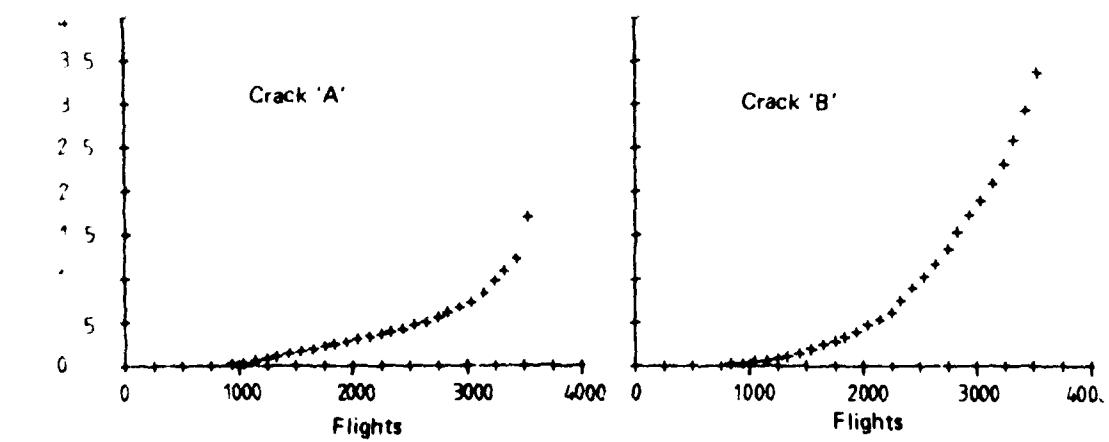
FIG. 8 (ii) CRACK PROPAGATION, TYPE (b) SPECIMENS



(a) Specimen GJ1ZB - 4 mm reamed hole, open



(b) Specimen GJ1Y - 4 mm reamed hole, adhesive coated, open



(c) Specimen GJ2A16 - 4 mm reamed hole, pressed-in rivet

FIG. 8 (i) CRACK PROPAGATION, TYPE (b) SPECIMENS

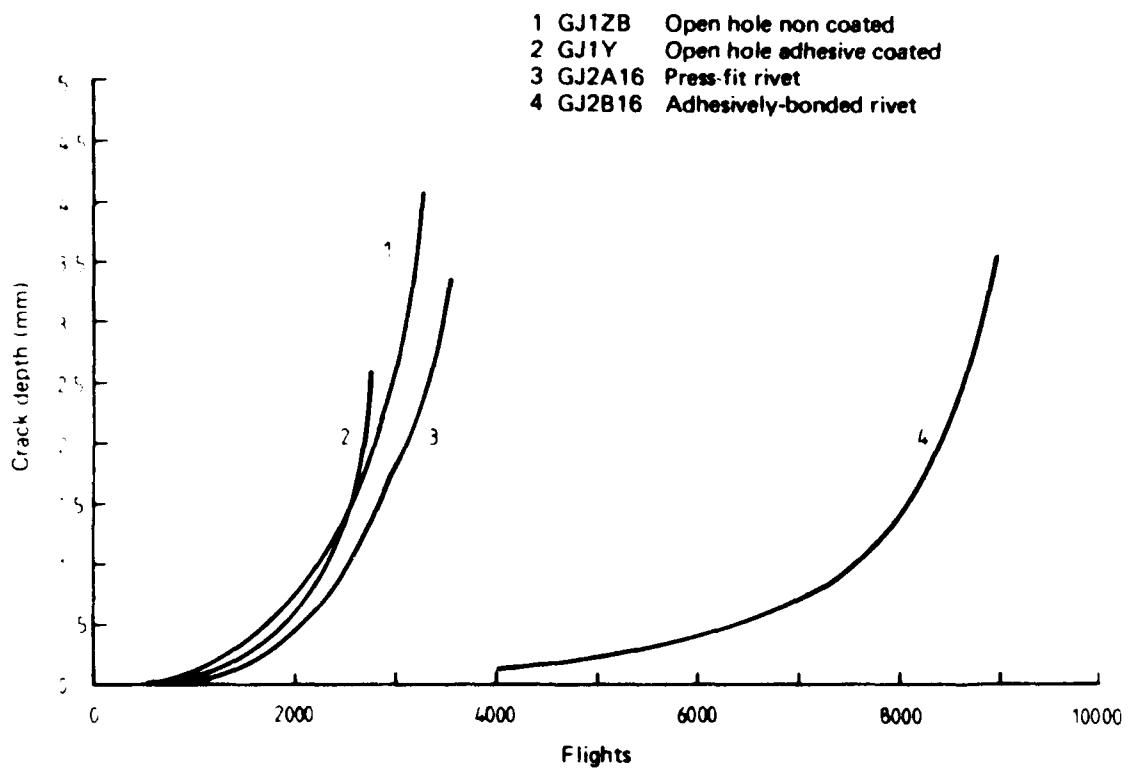
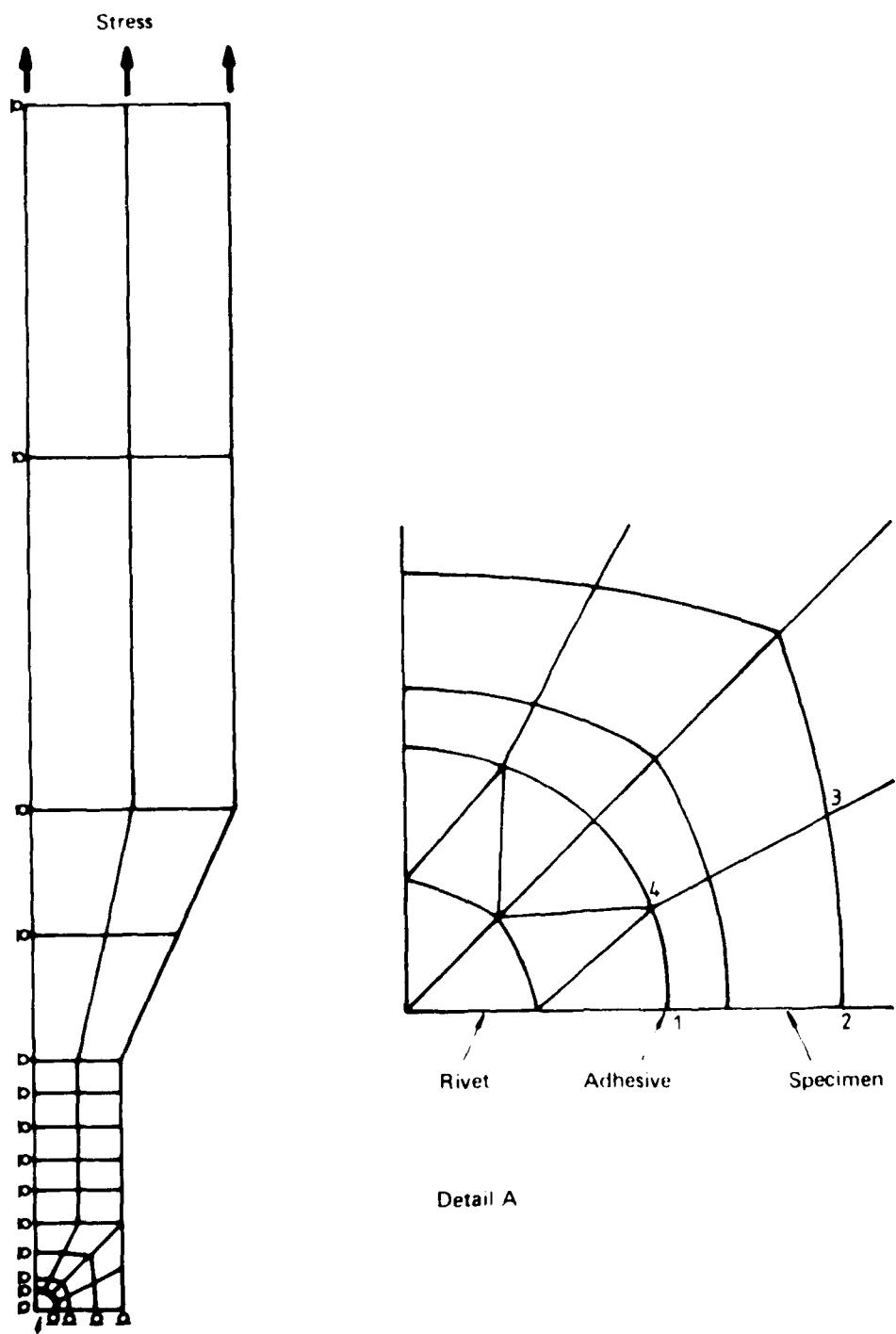


FIG. 9 CRACK PROPAGATION, TYPE (b) SPECIMENS



See Detail A

FIG. 10 (a) FINITE ELEMENT MESH FOR UNCRACKED SPECIMEN – BONDED RIVET

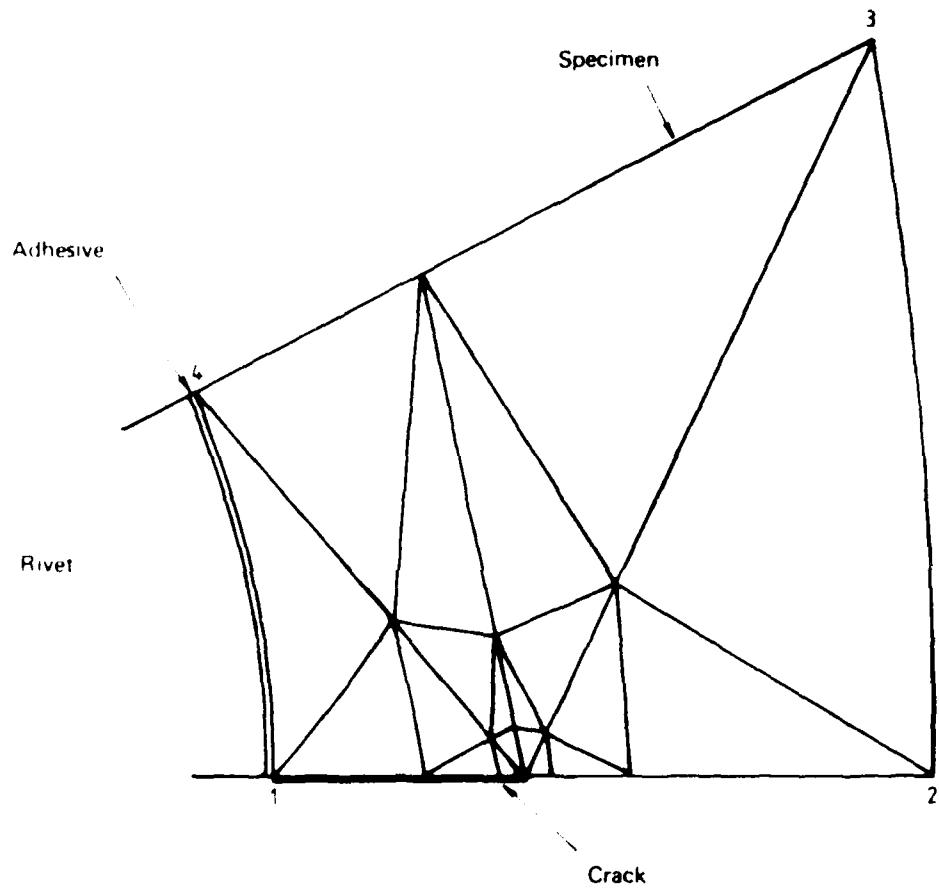


FIG. 10 (b) FINITE ELEMENT MESH FOR CRACKED SPECIMEN - BONDED RIVET  
(ONLY MESH IN REGION OF CRACK FOR QUARTER SECTION SHOWN)

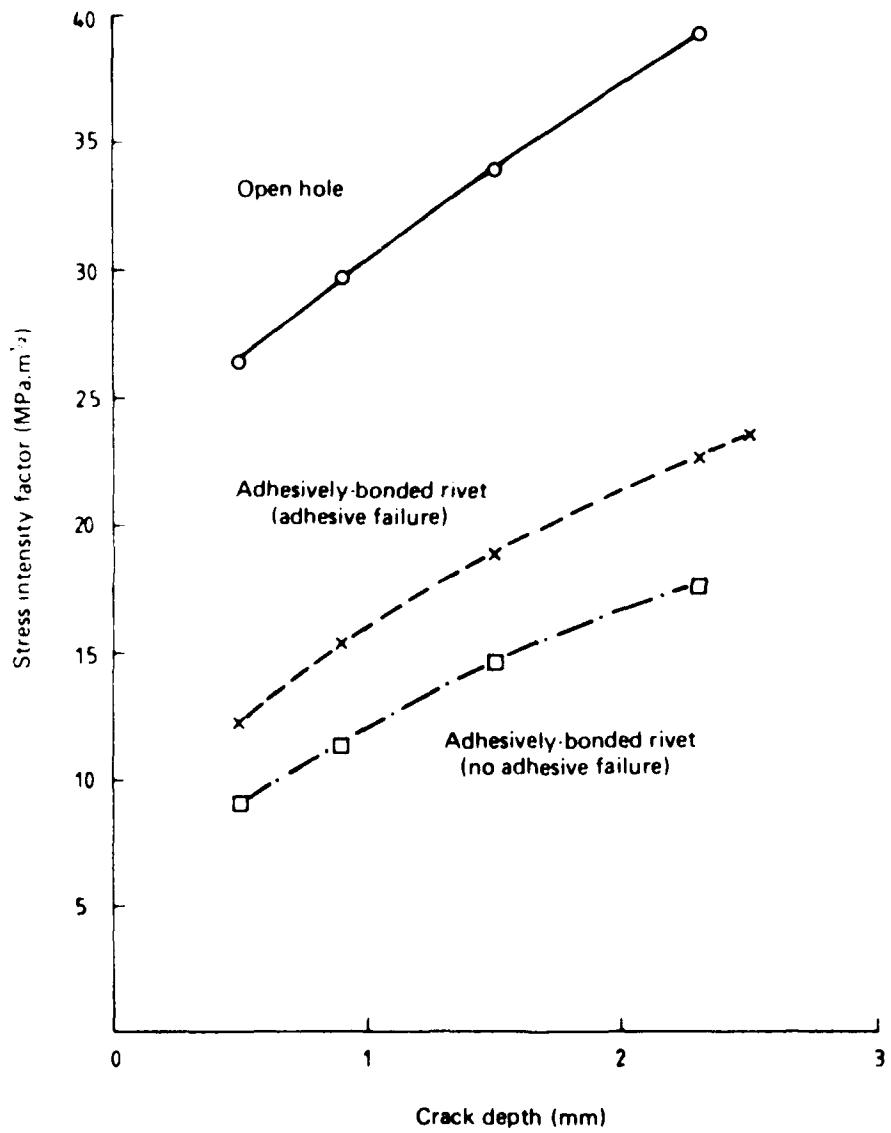


FIG. 11 STRESS INTENSITY FACTORS FOR HOLES WITH CRACKS ON BOTH SIDES

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16. Abstract Rivet holes are potential sites for fatigue crack initiation in aircraft structures. Several methods for improving the life of such details were investigated including coating the surface of the hole with adhesive, cold-expansion of the holes, the insertion of close-fit rivets and the use of adhesively-bonded rivets. Of the various techniques examined only that involving adhesively-bonded rivets provided any significant improvements in fatigue life. It resulted in a reduction in fatigue crack propagation rate of about 50% compared with that for specimens incorporating open holes. A finite element analysis indicated that adhesive bonding significantly reduces both the local stress concentration at the hole and the stress intensities at the crack tips, thus retarding crack			

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16 Abstract (Contd)

*initiation and reducing fatigue crack propagation rates. However, the effective reduction in stress intensity resulting from bonding (about 17%) is much less than the 50% predicted by the finite element analysis. This discrepancy is attributed mainly to shortcomings in the model for defining the characteristics and behaviour of the adhesive.*

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